

On the suitability of second-order accurate finite-volume solvers for the simulation of atmospheric boundary layer flow

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Abstract. The present work analyzes ~~the~~ quality and reliability of an important class of general-purpose, second-order accurate finite-volume (FV) solvers for the large-eddy simulation of a neutrally-stratified atmospheric boundary layer (ABL) flow. The analysis is carried out within the OpenFOAM[®] framework, which is based on a colocated grid arrangement. A series of open-channel flow simulations are ~~performed~~ carried out using a static Smagorinsky model for sub-grid scale momentum fluxes ~~and~~ in combination with an algebraic equilibrium wall-layer model. The sensitivity of the solution to variations in numerical parameters such as grid resolution (up to 160^3 control volumes), numerical solvers, and interpolation schemes for the discretization of ~~the nonlinear term is studied~~ nonlinear terms is evaluated and results are contrasted against those from a well established mixed pseudo-spectral–finite-difference code. Considered flow statistics include ~~the~~ mean streamwise velocity, resolved Reynolds ~~stress, turbulence intensities, skewness, kurtosis, spectra and spatial autocorrelations. The structure~~ of stresses, velocity skewness and kurtosis, velocity spectra and two-point autocorrelations. A quadrant analysis along with the examination of the conditionally-averaged flow field are performed to investigate the mechanisms responsible for momentum transfer in the flow ~~system is also discussed via a quadrant and a conditional-flow analysis. At the considered~~ It is found that, at the selected grid resolutions, the considered class of FV-based solvers yields a poorly correlated flow field and is not able to accurately capture the dominant mechanisms responsible for momentum transport in the ABL, ~~especially when using~~ linear interpolation schemes for the discretization of non-linear terms. The latter consist of sweeps. Specifically, the predicted flow field lacks the well-known sweep and ejection pairs organized side by side along the cross-stream direction, which are representative of a streamwise roll mode. This is especially true when using linear interpolation schemes for the discretization of nonlinear terms. This shortcoming leads to a misprediction of flow statistics that are relevant for ABL flow applications and to an enhanced sensitivity of the solution to variations in grid resolution, thus calling for future research aimed at reducing the impact of modeling and discretization errors.

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1 Introduction

An accurate prediction of atmospheric boundary layer (ABL) flows is of paramount importance across a wide range of fields and applications, including weather forecasting, complex terrain meteorology, agriculture, air quality modeling and wind energy
25 (Whiteman, 2000; Fernando, 2010; Calaf et al., 2010; Oke et al., 2017; Shaw et al., 2019).

Since the early work of Deardorff (1970), the large eddy simulation (LES) technique has spurred considerable insight on the fundamental dynamics of ABL flow over rough surfaces (Anderson and Meneveau, 2010; Salesky et al., 2017; Momen et al., 2018), over and within plant and urban canopies (Yue et al., 2007b; Bailey and Stoll, 2013; Pan et al., 2014; Tseng et al., 2006; Bou-Zeid et al., 2009; Giometto et al., 2017; Li and Bou-Zeid, 2019), and ABL flow for wind energy applications (Calaf et al.,
30 2010; Abkar and Porté-Agel, 2013; Stevens and Meneveau, 2017), amongst others.

The majority of past works has relied on fully or partially dealiased mixed pseudo-spectral–finite-difference (PSFD) solvers—the go-to approach for LES studies since the works of Moin et al. (1978) and Moeng (1984). Such solvers are known to yield accurate flow fields up to the LES cutoff frequency and to produce good results when used in conjunction with dynamic sub-grid scale (SGS) models (Germano et al., 1991; Lilly, 1992), ~~even when relying on a low-order finite-difference discretization~~
35 ~~in the vertical coordinate direction~~. However, single domain PSFD-based solvers are limited to regular domains, are not suitable for the simulation of non-periodic flows and sharp variations in the flow field such as shocks or ~~gas-solid interfaces~~, ~~and typically feature poor scaling~~ fluid-solid interfaces in boundary layer flows, and are typically difficult to parallelize owing to the global support of their spatial representation (see e.g. Margairaz et al., 2018). With the increasing need to account for complex geometries and multi-physics, several efforts have been devoted to the mitigation of the aforementioned limitations
40 (Fang et al., 2011; Li et al., 2016; Chester et al., 2007). ~~The solutions, however,~~ However, the solutions are often ad-hoc or validated only for specific applications, thus introducing a degree of uncertainty in model results that is hard to quantify and generalize.

There is hence a growing interest from the ABL community in LES solvers based on compact spatial schemes via structured or unstructured meshes (Orlandi, 2000; Ferziger and Peric, 2002). The parallelized large eddy simulation model (Raasch and
45 Schröter, 2001; Maronga et al., 2015) and the weather research and forecasting model (Skamarock et al., 2008; Powers et al., 2017) are prominent examples of said efforts. Both the approaches are based on a high-order finite-difference discretization, ~~and nonlinear terms are~~ with nonlinear terms approximated by using high-order upwind biased differencing schemes. The latter are suitable for LES in complex geometries with arbitrary grid stretching factors and outflow boundary conditions (Beaudan and Moin, 1994; Mittal and Moin, 1997), but are dissipative and do not strictly conserve energy. On the other hand, if central
50 schemes are used instead for the evaluation of nonlinear terms, no numerical dissipation is introduced, but truncation errors can have an overwhelming impact on the computed flow field (Ghosal, 1996; Kravchenko and Moin, 1997). These limitations typically result in a strong sensitivity of the solution to properties of the spatial discretization and numerical scheme (Meyers et al., 2006, 2007; Meyers and Sagaut, 2007; Vuorinen et al., 2014; Rezaeiravesh and Liefvendahl, 2018; Breuer, 1998; Montecchia et al., 2019). Further, truncation errors corrupt the high wavenumber range of the solution, restricting the ability
55 to adopt dynamic LES closure models which make use of information from the smallest resolved scales of motion to evaluate

the SGS diffusion (Germano et al., 1991). Notwithstanding these limitations, central schemes have been heavily employed in the past in both the geophysical and engineering flow communities, and are the de-facto standard in the wind engineering one, where most of the numerical simulations are carried out using second-order accurate finite-volume (FV) -based solvers (Stovall et al., 2010; Churchfield et al., 2010; Balogh et al., 2012; Churchfield et al., 2013; Shi and Yeo, 2016, 2017; García-Sánchez et al., 2017, 2018).

Motivated by the aforementioned needs, the present study aims at characterizing ~~the~~-quality and reliability of an important class of second-order accurate FV solvers for the LES of neutrally-stratified ABL flows. The analysis is conducted in the open-channel flow setup (no Coriolis acceleration) via the OpenFOAM[®] framework (Weller et al., 1998; De Villiers, 2006; Jasak et al., 2007). A suite of simulations is carried out varying physical and numerical parameters, including grid resolution (up to 65 160^3 control volumes), the numerical solver, and interpolation schemes for the discretization of the ~~non-linear-terms~~nonlinear term. Predictions from the FV solvers are contrasted against the results from the Albertson and Parlange (1999) PSFD code in terms of ~~first-, second-, and higher-order statistics, energy spectra, spatial~~flow statistics, including mean streamwise velocity, resolved Reynolds stresses, two-point velocity autocorrelations, and mechanisms supporting momentum transport. The end goal is to provide a more nuanced understanding of the capabilities of general-purpose, second order, FV-based solvers in 70 predicting ABL flow.

The work is organized as follows. Section 2 ~~briefly summarizes the governing equations, the numerical methods and the~~summarizes the setup of the problem, ~~along with a summary of the simulated cases~~the simulation database, and the post-processing procedure. Results are shown in §3 and conclusions are drawn in §4. A further discussion on the sensitivity of the solution to model constants, interpolation schemes, and numerical solvers is provided in the Appendix.

75 2 Methodology

2.1 Governing equations and numerical schemes

~~In the following, vector and index notations are used interchangeably, according to needs,~~We use index notation in a Cartesian reference system. The spatially-filtered Navier–Stokes equations are considered,

$$\frac{\partial u_i}{\partial t} + u_j \frac{\partial u_i}{\partial x_j} = -\frac{1}{\rho} \frac{\partial \tilde{p}}{\partial x_i} - \frac{\partial \tau_{ij}}{\partial x_j} - \frac{\partial \tau_{ij}^{SGS,dev}}{\partial x_j} - \frac{1}{\rho} \frac{\partial P}{\partial x_i}, \quad (1)$$

$$80 \quad \frac{\partial u_i}{\partial x_i} = 0, \quad (2)$$

$$\nabla \cdot \mathbf{u} = 0,$$

$$\frac{\partial \mathbf{u}}{\partial t} + \nabla \mathbf{u} \cdot \mathbf{u} = -\frac{1}{\rho} \nabla \tilde{p} + \nabla \cdot \boldsymbol{\tau} - \nabla \cdot \boldsymbol{\tau}^{SGS,dev} - \frac{1}{\rho} \nabla P,$$

where $\mathbf{u} = (u_x, u_y, u_z) = (u, v, w)$ where $u_i = (u, v, w)$ is the spatially-filtered velocity field along the streamwise (x), cross-stream (y) and vertical (z) coordinate directions, respectively, t is the time, ρ is the constant fluid density (Boussinesq approximation), $\tilde{p} \equiv p + \frac{1}{3}\tau_{kk}^{SGS}$ is the pressure term with an additional contribution from the sub-grid kinetic energy ($\frac{1}{2}\tau_{kk}^{SGS}$), $\tilde{p} \equiv p + \frac{1}{3}\tau_{kk}^{SGS}$ is a modified pressure term, τ_{ij} is the filtered viscous stress tensor, $\tau_{ij}^{SGS,dev}$ and $\tau_{ij}^{SGS,dev}$ is the deviatoric part of the SGS stress tensor. In addition, the term $-\frac{1}{\rho}\nabla P$ is a pressure gradient, here assumed to be constant and uniform, responsible for $-\frac{1}{\rho}\frac{\partial P}{\partial x_i}$ is an imposed constant pressure gradient driving the flow. The filtered spatially-filtered viscous tensor is $\tau = -2\nu \mathbf{S} \tau_{ij} = -2\nu S_{ij}$, where $\nu = \text{const}$ is the kinematic viscosity of the Newtonian fluid and $\mathbf{S} S_{ij}$ is the resolved (in the LES sense) rate of strain tensor. For the SGS stress tensor, the static Smagorinsky model is used,

$$\tau_{ij}^{SGS,dev} = -2\nu^{SGS} \mathbf{S} S_{ij} = -2(C_{SS}\Delta)^2 |\mathbf{S}| \mathbf{S} S_{ij}, \quad (3)$$

where ν^{SGS} is the SGS eddy viscosity, $C_S C_S$ is the Smagorinsky coefficient (Smagorinsky, 1963), $\Delta = (\Delta x \Delta y \Delta z)^{1/3}$ is a local length scale based on the volume of the computational cell (Scotti et al., 1993), and $|\mathbf{S}| = \sqrt{2\mathbf{S} \cdot \mathbf{S}}$ $|\mathbf{S}| = \sqrt{2S_{ii}S_{ii}}$ quantifies the magnitude of the rate of strain. In the present work, $C_S = 0.1$ $C_S = 0.1$, unless otherwise specified. Note that dynamic Smagorinsky models are preferred to the static one for the LES of ABL flows (Germano et al., 1991; Lilly, 1992; Meneveau et al., 1996; Porté-Agel, 2004; Bou-Zeid et al., 2005). Dynamic models evaluate SGS stresses via first-principles-based constraints, feature improved dissipation properties when compared to the static Smagorinsky one (especially in the vicinity of solid boundaries), and are free of explicit modeling parameters. The choice made in the present study is motivated by problematics encountered when using the available dynamic Lagrangian model in preliminary tests. However, while SGS dissipation plays a crucial role in PSFD solvers, truncation errors may overshadow SGS stress contributions in the second-order FV-based ones (Kravchenko and Moin, 1997). The static Smagorinsky SGS model used herein might hence perform similarly to dynamic SGS models for the considered flow setup. This conjecture is supported by the results of Majander and Siikonen (2002).

The large scale separation between near-surface and outer-layer energy containing ABL motions poses stringent resolution requirements to numerical modelers, if all the energy containing motions have to be resolved. To reduce the computational cost of such simulations, the near-surface region is typically bypassed, and a phenomenological wall-layer model is leveraged instead to account for the impact of near-wall (inner-layer) dynamics on the outer-layer flow (Piomelli, 2008; Bose and Park, 2018). This approach is referred to as wall-modeled large eddy simulation (WMLES) and is used herein. An algebraic wall-layer model for surfaces in fully rough aerodynamic regime was implemented, based on the logarithmic equilibrium assumption, i.e.,

$$|\mathbf{u}_{\tilde{u}}| = \frac{u_*}{\kappa} \ln\left(\frac{z}{z_0}\right), \quad (4)$$

where $|\mathbf{u}| \equiv \sqrt{u^2 + v^2}$ $|\tilde{\mathbf{u}}| \equiv \sqrt{u^2 + v^2}$ is the norm of the velocity at a certain distance from the ground level, u_* is the friction velocity (see Sub-Section 2.2 for details), κ is the von Kármán constant, z is the distance from the ground level and

115 z_0 is the so-called aerodynamic roughness length, a length-scale used to quantify the drag of the underlying surface. In this work we define, the values $\kappa = 0.41$ and $z_0 = 0.1$ m are set. The kinematic wall shear stress is assumed to be proportional to the local velocity gradient (Boussinesq hypothesis),

$$\tau_{iz,w} = (\nu + \nu_{tt}) \frac{\partial u_i}{\partial z} \Big|_{w}, \quad i = x, y, \quad (5)$$

with ν_t being ν_t the total eddy viscosity. Employing the no-slip condition for the velocity field, the standard FV approximation of the shear stress at the wall gives (Mukha et al., 2019)

$$120 \quad \tau_{iz,w} = (\nu + \nu_{tt}) f \frac{u_{i,c}}{\Delta z} f, \quad i = x, y, \quad (6)$$

where the subscript f is used to denote the evaluation at the center of the wall face, the subscript c denotes the evaluation at the center of the wall-adjacent cell and Δz is the distance from the wall. From the logarithmic law (Eq. 4) evaluated at the first cell-center, one can write $u_* = \kappa |\mathbf{u}|_c / \ln(\frac{\Delta z}{z_0})$, $u_* = \kappa |\tilde{\mathbf{u}}|_c / \ln(\frac{\Delta z}{z_0})$. Using the definition of friction velocity $u_* = \sqrt{\tau_w^2}$, where $\tau_w = \sqrt{\tau_w^2}$, where τ_w is the magnitude of the kinematic wall shear stress vector, along with Eq. 5 and rearranging terms, and rearranging, the total eddy viscosity at the wall can be written as

$$125 \quad \nu_{t,f} = \left(\frac{\kappa |\mathbf{u}|_c}{\ln\left(\frac{\Delta z}{z_0}\right)} \frac{\kappa |\tilde{\mathbf{u}}|_c}{\ln\left(\frac{\Delta z}{z_0}\right)} \right)^2 \frac{\Delta z}{|\mathbf{u}|_c} \frac{\Delta z}{|\tilde{\mathbf{u}}|_c} - \nu, \quad (7)$$

which is the formulation implemented herein. Note that $\nu + \nu_t \approx \nu_t$ in boundary layer flows in fully rough aerodynamic regime, so that ν could be neglected without loss of accuracy.

In the OpenFOAM® framework present work, the computational grid is collocated, being the collocated grid arrangement the only one available within the OpenFOAM® framework. Although Note that, although advantageous in complex domains when compared to staggered grids (Ferziger and Peric, 2002), the collocated arrangement is known to cause difficulties with pressure-velocity coupling, hence requiring specific procedures to avoid oscillations in the solution. The OpenFOAM® offers the standard Rhie-Chow correction (Rhie and Chow, 1983) is here adopted, which is known to negatively affect the energy-conservation properties of central schemes (Ferziger and Peric, 2002). In addition, when approximating the integrals over the surfaces bounding each control volume (as a consequence of the Gauss divergence theorem), the unknowns are evaluated at face-centers and are assumed to be constant at each face, yielding an overall second-order spatial accuracy (Churchfield et al., 2010). Since the divergence form of the convective term is used in combination with a low-order scheme over a non-staggered grid, the solution is inherently unstable (Kravchenko and Moin, 1997). The present work makes use of the linear and the QUICK interpolation schemes (Ferziger and Peric, 2002) to evaluate the unknowns at face-centers (more details are provided in § 2.2). The numerical solver combines is based on the PISO algorithm (Issa, 1985) for the pressure-velocity calculation and on an implicit Adams–Moulton scheme for time integration (Ferziger and Peric, 2002). The In the Appendix A2, the

performances of an alternative solver characterized-by-with a Runge–Kutta time-advancement scheme and a projection method for the pressure-velocity coupling (Vuorinen et al., 2014) are also-analyzed-in-the-Appendix-A2-analyzed.

2.2 Problem setup

145 A series of WMLES of ABL flow (open-channel flow setup) is performed. Tests are carried out in the domain $[0, L_x] \times [0, L_y] \times [0, L_z]$ with $L_x = 2\pi h$, $L_y = \frac{4}{3}\pi h$, $L_z = h$, where $h = 1000$ m denotes the width of the open channel. Symmetry is imposed at the top of the computational domain, no-slip applies at the lower surface and periodic boundary conditions are enforced along each side. A kinematic pressure gradient term $-\frac{1}{\rho}\partial P/\partial x = 1 \text{ m/s}^2$ $-\frac{1}{\rho}\partial P/\partial x = 1 \text{ m/s}^2$ drives the flow along the x coordinate direction, yielding $u_* = 1$ m/s. The kinematic viscosity is set to a nominal value of $10^{-7} \text{ m}^2/\text{s}$, which results in an essentially
150 inviscid flow.

The computational mesh is Cartesian, with a uniform stencil along each direction. Three simulations are run, over 64^3 , 128^3 and 160^3 control volumes, with the linear interpolation scheme for the evaluation of the unknowns at the face-centers (simulations FV64, FV128 and FV160, respectively). Three additional simulations are run, over-at the same grid resolutions, with the linear scheme for the approximation of every term except for the nonlinear one, for which the QUICK scheme is
155 used instead (simulations FV64*, FV128* and FV160*). The cases span different grid resolutions at the same aspect ratio $\Delta x/\Delta z = 2\pi$. Note that the chosen grid resolutions are in line with those typically used in studies of ABL flows-using-a-flow-with-the pseudo-spectral approach (see, e.g., Salesky et al., 2017). All the calculations satisfy the Courant–Friedrichs–Lewy (CFL) condition $Co \lesssim 0.1$, where $Co = C \lesssim 0.1$, where C is the Courant number. Runs are initialized from a fully developed open-channel flow simulation in statistically steady state (dynamic equilibrium), and time integration is carried out for 100 eddy
160 turnover times, where the eddy turnover time is defined as h/u_τ . Flow statistics are the result of an averaging procedure in-over the horizontal plane of statistical homogeneity of turbulence (xy) and in time over the last 60 eddy turnover times. The procedure yields well converged statistics throughout the considered cases. In the following, the horizontal and temporal averaging operation is denoted by $\langle \cdot \rangle$.

Results-The results from the present study are contrasted against the corresponding ones from the Albertson and Parlange
165 (1999) mixed PSFD code (simulations PSFD64, PSFD128 and PSFD160). The code is based on an explicit second-order accurate Adams–Bashforth scheme for time integration and on a fractional-step method for solving the system of equations. Simulations from the PSFD solver are carried out using a static Smagorinsky SGS model with $C_s = 0.1$, a rough wall-layer model with $z_0 = 0.1$ m, and $Co \lesssim 0.1$. A summary of the runs is given in Tab. 1 along with the acronyms used in this study.

170 3 Results

This Section is devoted to the analysis of velocity central moments (§3.1), spectra and spatial autocorrelations (§3.2), and momentum transfer mechanisms (§3.3).

Table 1. Tabulated list of cases.

simulation	FV64	FV128	FV160	FV64*	FV128*	FV160*	PSFD64	PSFD128	PSFD160
grid resolution $N_x \times N_y \times N_z$	64^3	128^3	160^3	64^3	128^3	160^3	64^3	128^3	160^3
numerical solver	FV	FV	FV	FV + QUICK	FV + QUICK	FV + QUICK	PSFD	PSFD	PSFD

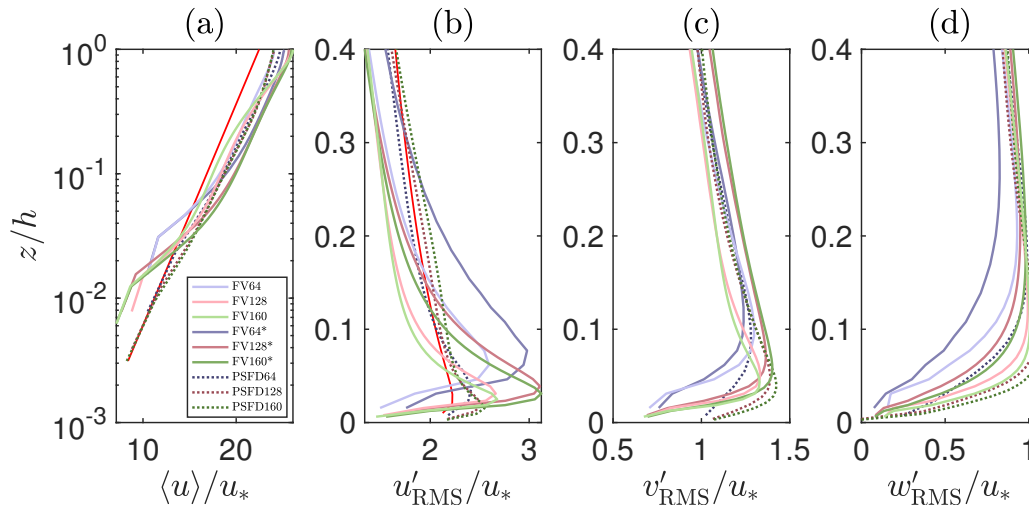


Figure 1. Vertical structure of mean streamwise velocity $\langle u \rangle / u_*$ (a), streamwise velocity RMS u'_{RMS} / u_* (b), cross-stream velocity RMS v'_{RMS} / u_* (c) and vertical velocity RMS w'_{RMS} / u_* (d). The red line in (a) denotes the reference logarithmic profile and the red line in (b) is a reference profile from Hultmark et al. (2013).

3.1 Mean velocity, Reynolds stresses, and higher order statistics

Figure 1 shows first and second order statistics for all the considered cases. The mean streamwise velocity is shown in Fig. 1(a),
175 in a comparison with the phenomenological logarithmic-layer profile. The velocity at the first two cell-centers off the wall is consistently underpredicted, whereas a positive log-layer mismatch (LLM) is observed in the bulk of the flow (Kawai and Larsson, 2012). The LLM is particularly pronounced for the cases using the QUICK interpolation scheme. This behavior could have been anticipated, as the wall shear stress is evaluated using the instantaneous horizontal velocity at the first cell-center off the wall. A number of procedures ~~have has~~ been proposed to alleviate the LLM, including modifying the SGS stress model
180 in the near-wall region (Sullivan et al., 1994; Porté-Agel et al., 2000; Chow et al., 2005; Wu and Meyers, 2013), shifting the matching location further away from the wall (Kawai and Larsson, 2012), and carrying out a local horizontal/temporal filtering operation (Bou-Zeid et al., 2005; Xiang et al., 2017). In preliminary runs, ~~we implemented~~ the approach of Kawai and Larsson (2012) was implemented in an attempt to alleviate the LLM; ~~but~~. However, no apparent improvement was observed and the solution became very sensitivity to grid resolution and matching location. This finding suggests that alternative procedures

Table 2. Relative error on the turbulence intensities $\|u' - u'_{\text{ref}}\|_{L^2} / \|u'_{\text{ref}}\|_{L^2} - \|u'_{\text{RMS}} - u'_{\text{RMS,ref}}\|_{L^2} / \|u'_{\text{RMS,ref}}\|_{L^2}$ w.r.t. the reference profile from Hultmark et al. (2013), in the interval $z_0/h \leq z/h \leq 0.4$.

simulation	FV64	FV128	FV160	FV64*	FV128*	FV160*	PSFD64	PSFD
relative error on $u' - u'_{\text{RMS}}$	0.2259-0.23	0.1696-0.17	0.1722-0.17	0.2808-0.28	0.2036-0.20	0.1965-0.20	0.1050-0.10	0.0489

185 might need to be devised to overcome the LLM in ABL flow simulations when using the considered class of FV solvers. Note that profiles from the PSFD solver also feature a positive LLM, in spite of a spatial, low-pass filtering operation that is carried out on the horizontal velocity field before evaluating the surface shear stress (Bou-Zeid et al., 2005).

The vertical structure of turbulence intensities is also shown in Fig. 1, where $(\cdot)'_{\text{RMS}}$ denotes the root mean square (RMS) of the fluctuations. Profiles from the FV-based solver start off relatively slow at the wall when compared to those from
190 the PSFD-based solver and to the reference profile from Hultmark et al. (2013). This behavior is due to a combination of SGS and discretization errors, which damp the energy of high-wavenumber modes and whose accurate quantification remains an open challenge in LES (see e.g., Meyers et al., 2006; Meyers and Sagaut, 2007; Meyers et al., 2007). Further aloft, $u' - (u' u'_{\text{RMS}})$ (w'_{RMS}) features relatively stronger (weaker) peak values when compared to the corresponding PSFD profile, the overprediction (underprediction) being more apparent in the simulations with the QUICK scheme. The overshoot in the peak of $u' - u'_{\text{RMS}}$ is a
195 well-known problem of FV-based WMLES (Bae et al., 2018). Lack of energy redistribution via pressure fluctuation from shear generated $u' - u'_{\text{RMS}}$ and w'_{RMS} is the root cause of said behavior, and possible mitigation strategies include allowing for wall transpiration (Bose and Moin, 2014). Grid refinement shifts the velocity RMS peaks closer to the surface and increases the magnitude of velocity RMS therein, but lead to no improvement in the $\max(u') - \max(u'_{\text{RMS}})$ and only marginally improves the estimation of the $\max(w') - \max(w'_{\text{RMS}})$. A quantitative measure of the relative error on $u' - u'_{\text{RMS}}$ with respect to
200 the reference profile $u'_{\text{RMS,ref}}$ from Hultmark et al. (2013) in the $z_0/h \leq z/h \leq 0.4$ interval is shown in Tab. 2. It is clear that the PSFD solver performs best and that the FV solution with QUICK performs worst, but what's more important is that The FV-based solver performs worse than the PSFD-based one, and the convergence is not monotonic. Non-monotonic convergence is relatively common in LES at relatively coarse resolutions and is due to the interaction between discretization and modeling errors, whose impact on the solution cannot be a-priori quantified (Meyers et al., 2007).

205 Skewness and kurtosis of the streamwise velocity (S_{uu} and K_{uu} , respectively) are shown in Fig. 2. Profiles The profiles of S_{uu} obtained with the FV-based solver in combination with and the QUICK scheme and those from the simulations carried out as well as those obtained with the PSFD-based solver are in good agreement with experimental results from Monty et al. (2009), which are here taken as a reference. The FV-based solver, on On the contrary, the FV-based solver overpredicts S_{uu} when the linear interpolation scheme is used, with the skewness remaining positive throughout the whole extent of the surface
210 layer. Note that a positive skewness of streamwise velocity represents a flow field where negative fluctuations are more likely to happen than the correspondent corresponding positive ones. The kurtosis obtained with the FV-based solver is consistently overpredicted, representing a flow field populated by a greater number of extreme events. Again, profiles from all cases feature

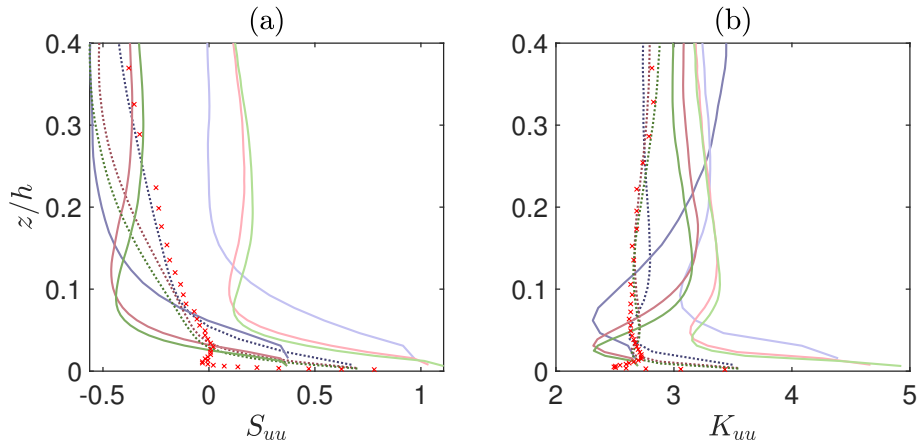


Figure 2. Vertical structure of skewness of streamwise velocity (a) and kurtosis of streamwise velocity (b). Lines are defined in Fig. 1. The red x-marks denote the measurements from Monty et al. (2009), digitalized by the authors.

Table 3. Relative error on skewness $\|S_{uu} - S_{uu,meas}\|_{L^2} / \|S_{uu,meas}\|_{L^2}$ and kurtosis $\|K_{uu} - K_{uu,meas}\|_{L^2} / \|K_{uu,meas}\|_{L^2}$ w.r.t. the measurements from Monty et al. (2009), in the interval $z_0/h \leq z/h \leq 0.4$.

simulation	FV64	FV128	FV160	FV64*	FV128*	FV160*	PSFD64	PSFD128
relative error on S_{uu}	1.6339 <u>1.63</u>	1.7656 <u>1.77</u>	1.8142 <u>1.81</u>	0.8577 <u>0.86</u>	0.7495 <u>0.75</u>	0.7083 <u>0.71</u>	0.5122 <u>0.51</u>	0.5719 <u>0.57</u>
relative error on K_{uu}	0.2832 <u>0.28</u>	0.2546 <u>0.25</u>	0.2538 <u>0.25</u>	0.2271 <u>0.23</u>	0.1644 <u>0.16</u>	0.1535 <u>0.15</u>	0.1093 <u>0.11</u>	0.0459 <u>0.05</u>

[feature](#) a non-monotonic convergence to the reference ones, as shown in Tab. 3, where the relative error on skewness and kurtosis with respect to the measurements from Monty et al. (2009) is reported, in the interval $z_0/h \leq z/h \leq 0.4$.

215 3.2 Spectra and autocorrelations

One-dimensional spectra of streamwise velocity fluctuations (E_{uu}) ~~for each of the considered cases~~ are shown in Fig. 3(a). ~~Profiles~~ [The profiles](#) are contrasted against the phenomenological production range and inertial sub-range power-law profiles (k^{-1} and $k^{-5/3}$, respectively). Predictions from the PSFD-based solver feature a relatively good agreement with the phenomenological power-law profile, ~~especially at the two highest grid resolutions, which feature a slope of -1.2 for especially~~ [at high grid resolution. For example](#), the cases PSFD128 and PSFD160 [exhibit a slope of \$-1.2\$ in the production range](#) (here defined as $k_x z < 1$). Profiles from the FV-based solver, on the contrary, exhibit strong sensitivity to grid resolution and are unable to capture the expected power-law behavior. In the production range, velocity spectra from the FV solver start off relatively shallow at small ~~wavenumbers, especially wavenumber, especially~~ [when using the linear scheme. A narrow band can be identified](#) where ~~$E_{uu} \sim (k_x z)^{-1}$ can then be identified~~ [where \$E_{uu} \sim \(k_x z\)^{-1}\$](#) , followed by a rapid decay in energy density. ~~The~~

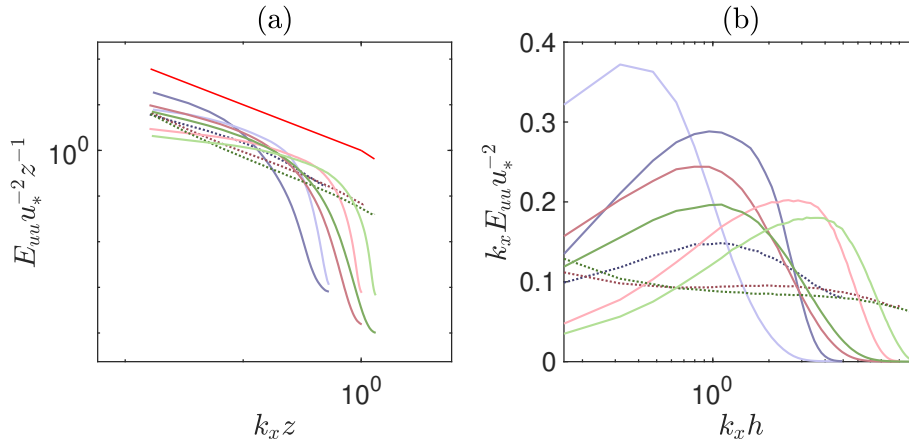


Figure 3. (a) Normalized one-dimensional spectrum spectra of streamwise velocity at height $z/h \approx 0.1$. Solid The solid red line $(k_x z)^{-1}$ in depicts the $(k_x z)^{-1}$ production range and $(k_x z)^{-5/3}$ in the inertial sub-range scaling. All the other lines as in Fig. 1. (b) Premultiplied one-dimensional spectrum spectra of streamwise velocity at $z/h \approx 0.1$.

225 decay is especially density—the decay being particularly pronounced when using the QUICK interpolation scheme, because of the associated numerical dissipation.

Overall, the energy density in the production range and in the inertial sub-range are is not well captured by the FV-based solver and grid refinement does not help circumvent this limitation, at least at the considered resolutions. The authors note that this fact might limit the use of dynamic procedures based on the Germano et al. (1991) identity.

230 A further characterization of the energy distribution in the wavenumber space is given in Fig. 3(b), where premultiplied velocity spectra $k_x E_{uu} u_*^{-2}$ are shown. The usual reason for considering this quantity these quantities is to create a plot in semi-log scale where equal areas under the profiles correspond to equal energy. In addition, premultiplied spectra provide information on the coherence of the flow, in particular on the so-called large and very large scale motions (LSMs and VSLMs, respectively). These structures are responsible for carrying more than half of the kinetic energy and Reynolds shear stress and are a persistent
 235 feature of the surface and outer layers of both aerodynamically smooth and rough walls (Kim and Adrian, 1999; Balakumar and Adrian, 2007; Monty et al., 2007; Hutchins and Marusic, 2007; Fang and Porté-Agel, 2015). The current domain is of modest dimensions and should only be is able to accommodate only LSMs (Lozano-Durán and Jiménez, 2014), which are identified in premultiplied spectra by a local maximum at the streamwise wavenumber $k_x/h \approx 1$. The location of the peaks from the FV-based solver with linear interpolation scheme shifts toward higher wavenumbers wavenumber with grid refinement, with
 240 a maximum at $k_x/h \approx 4$ for the FV160 case. This fact signals a flow field where the streamwise extent of energetic modes (a.k.a., coherent structures) reduces as the grid is refined. The On the contrary, the FV-based solver in combination with the QUICK scheme on the contrary, predicts the peak in premultiplied energy density at the expected wavenumber, hence suggesting that this approach is able to capture LSMs. The PSFD-based solver features a peak at the expected wavenumber ($k_x = 1$) only at the lowest resolution (PSFD64). Profiles from the higher resolution cases feature high energy densities at the

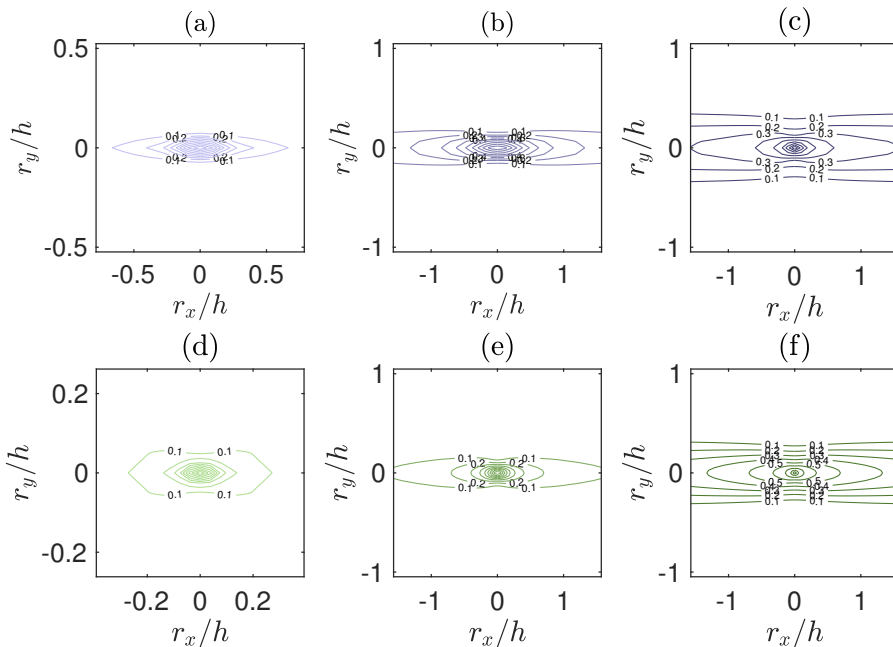


Figure 4. Contours of two-dimensional spatial autocorrelation of streamwise velocity at height $z/h \approx 0.1$, from the simulations FV64 (a), FV64* (b), PSFD64 (c), FV160 (d), FV160* (e) and PSFD160 (f). Contour levels from 0.1 to 0.9 with increments of 0.1.

245 lowest ~~wavenumbers~~wavenumber, highlighting an artificial “periodization” of energy-containing structures in the streamwise direction. This behavior is linked to the limited horizontal extent of the computational domain. The authors have indeed verified that a ~~larged~~-larger domain (twice as large along each horizontal direction) enables to capture LSM with the PSFD solver at resolutions matching ~~that-the one~~ of the PSFD128 case (not shown). A corresponding single run ~~of-the-FV-solver-was-also~~ ~~carried-out-the-was~~ carried out with the FV solver over the said larger domain and premultiplied spectra were found to be

250 in good agreement with those presented herein, supporting the ~~working~~-conjecture that the proposed domain size suffice to capture the range of variability of FV solvers for the problem under consideration.

To gain better insight on the spatial coherence of the flow field, the contour lines of the two-dimensional autocorrelation of the streamwise velocity R_{uu}^{2D} in the xy plane are shown in Fig. 4. The ~~$R_{uu} = 0.1$~~ - $R_{uu}^{2D} = 0.1$ contour is often used to identify the boundaries of coherent structures populating the flow field. The contours from the FV-based solver with linear

255 scheme (Figs. 4,a,d) are representative of a poorly correlated flow field, with a streamwise extent of the ~~$R_{uu} = 0.1$ contour~~ ~~extending-~~ $R_{uu}^{2D} = 0.1$ contour of $0.5h$ and $0.1h$ ~~in-along~~ along the streamwise and ~~spanwise-cross-stream~~ spanwise-cross-stream directions, respectively. On the contrary, the contours from the FV-based solver with QUICK scheme (Fig. 4,b,e) depict a flow field characterized by larger spatial autocorrelation, in line with results from the PSFD-based solver.

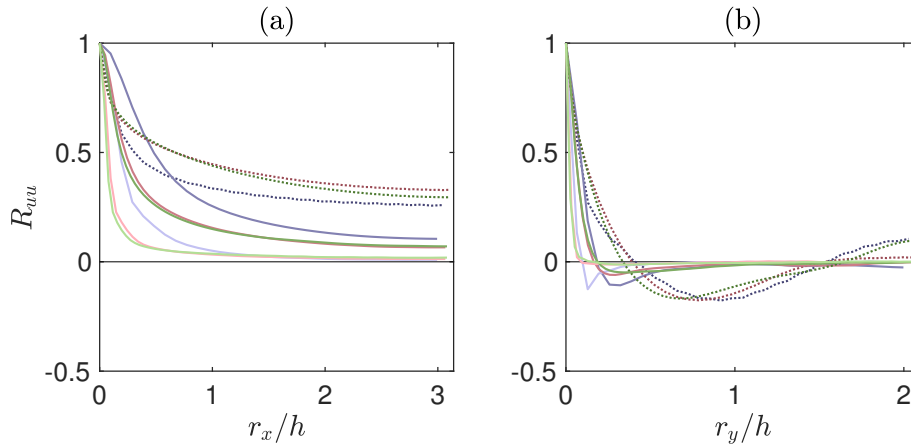


Figure 5. One-dimensional spatial autocorrelation of streamwise velocity at height $z/h \approx 0.1$, along the streamwise direction (a) and along the cross-stream direction (b). Lines as in Fig. 1.

Note that the flow statistics presented ~~before~~ above should not be impacted by the fact that the current domain size prevents some of the contour lines (simulations FV64*, FV160*, PSFD64, PSFD160) from closing (~~Lozano-Durán and Jiménez, 2014~~), as discussed in Lozano-Durán and Jiménez (2014).

The one-dimensional spatial autocorrelation (R_{uu}), shown in Fig. 5 along the streamwise and cross-stream directions, further corroborates the above findings. From Fig. 5(a) it is apparent that the extension of the selected domain does not enable the flow to become completely uncorrelated in the streamwise direction for the PSFD solver and for the FV solver using QUICK: R_{uu} remains finite in the available r_x/h range across resolutions. ~~Profiles~~ On the other hand, profiles from the FV-based solver ~~with using a linear interpolation, on the other hand, decay rapidly using the linear interpolation rapidly decay~~ towards zero.

Along the cross-stream direction (Fig. 5,b), profiles from the PSFD-based solver feature the expected negative lobes, highlighting the presence of high- and low-momentum streamwise-elongated streaks flanking each others in the said direction. This behavior is in line with findings from previous studies on the coherence of wall-bounded turbulence and with standard turbulence theory. Profiles from the FV-based solver exhibit a similar profile, albeit featuring a more rapid decay and less prominent negative lobes, especially for the high-resolution cases using the linear interpolation scheme.

A quantitative measure of the coherence of the flow field is provided in Tab. 4, where the integral lengths $\Lambda_{r_x,u}$ and $\Lambda_{r_y,u}$ are reported for all the considered cases and ~~for the~~ compared against direct numerical simulations of a channel flow at $Re_\tau = 2000$ from Sillero et al. (2014). The integral lengths in Tab. 4 are evaluated at $z/h \approx 0.15$, since the data from Sillero et al. (2014) ~~were~~ are available at this height. Although $\Lambda_{r_x,u}$ might not be meaningful across the considered cases, owing to the lack of a zero crossing of the autocorrelation function, it is apparent that the FV-based solver underestimates the integral lengths when compared to the PSFD cases and the reference DNS values, especially when the linear interpolation scheme is used.

Instantaneous snapshots of streamwise velocity fluctuations over a horizontal plane ~~confirm~~ said ~~support~~ the above findings (see Fig. 6). Artificially-periodized, streamwise-elongated bulges of uniform high and low momentum are indeed apparent in

Table 4. Integral lengths at height $z/h \approx 0.15$.

simulation	FV64	FV128	FV160	FV64*	FV128*	FV160*	PSFD64	PSFD128	PSFD160
$\Lambda_{r_x, u}/h$	0.2320-0.23	0.1203-0.12	0.1151-0.11	0.8246-0.82	0.5879-0.59	0.5930-0.59	1.2810-1.28	1.5009-1.50	1.4523-1.45
$\Lambda_{r_y, u}/h$	0.0379-0.04	0.0293-0.03	0.0277-0.03	0.0967-0.10	0.0828-0.08	0.0828-0.08	0.1436-0.14	0.1546-0.15	0.1433-0.14

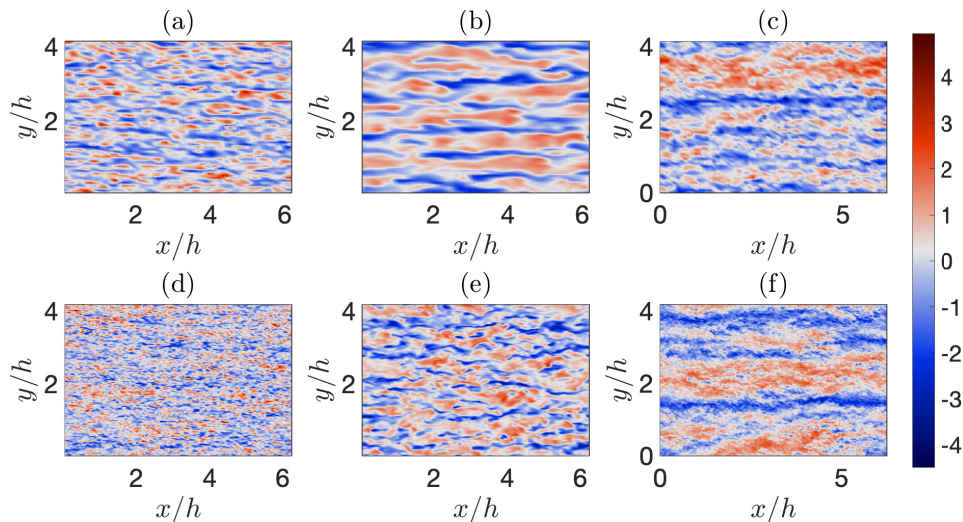


Figure 6. Instantaneous snapshots of normalized streamwise velocity fluctuations at $z/h \approx 0.1$ from the simulations FV64 (a), FV64* (b), PSFD64 (c), FV160 (d), FV160* (e) and PSFD160 (f). The normalized velocity fluctuation is defined as $(u - \langle u \rangle_{xy}) / u''(u - \langle u \rangle_{xy}) / u_*$, where averages (and fluctuations therefrom) are evaluated in space over the selected horizontal plane.

280 the snapshots from the PSFD-based solver (Fig. 6,c,f). ~~The~~ On the contrary, the instantaneous streamwise velocity field from the FV solver ~~, on the contrary, appears to be is~~ populated by smaller regions of uniform momentum, especially when using the linear scheme, and the size of energetic structures diminishes with increasing grid resolution (see, e.g., Figs. 6,a,d).

3.3 Momentum transfer mechanisms

This ~~section~~ Section is devoted to the analysis of momentum transfer mechanisms in the ABL, with a focus on quadrant analysis
285 (Lu and Willmarth, 1973) and on statistics of conditionally-averaged flow fields.

The quadrant hole analysis is a technique based on the decomposition of the velocity fluctuations into four quadrants: the first and third quadrants, *outward interactions* ($u' > 0, w' > 0$) and *inward interactions* ($u' < 0, w' < 0$), respectively, are negative contributions to the momentum flux, whereas the second and fourth quadrants, a.k.a. *ejections* of low-speed fluid outward from the wall ($u' < 0, w' > 0$) and *sweeps* of high-speed fluid toward the wall ($u' > 0, w' < 0$), represent positive contributions. A

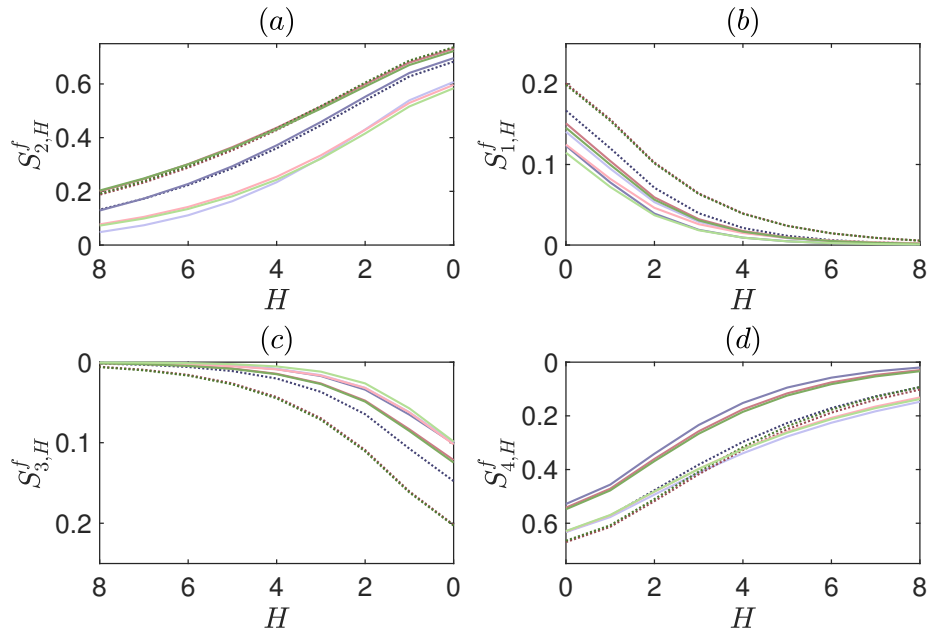


Figure 7. Stress fractions at $z/h \approx 0.1$. The profiles are normalized so that the sum of the stress fractions for $H = 0$ is unity across the cases. Lines are defined in Fig. 1.

290 range of flow statistics can be defined based on ~~the said this~~ decomposition and used to provide insight on the mechanisms supporting momentum transfer in the ABL.

Figure 7 features the quadrant-hole analysis, where the notation is the same as in Yue et al. (2007a), with H being the hole size, $S_{i,H}$ the resolved Reynolds shear stress contribution to the i -th quadrant at hole size H , and $S_{i,H}^f$ is the ~~correspondent~~ corresponding quadrant fraction. Stress fractions are presented for values of the hole size H ranging from 0 to 8, where
 295 larger hole sizes correspond to contributions to the resolved Reynolds shear stress from more extreme events. Clearly, the FV-based solver with the linear scheme underpredicts ejections (Fig. 7,a), outward interactions (Fig. 7,b) and inward interactions (Fig. 7,c), and overpredicts sweeps at large hole size H (Fig. 7,d). ~~The QUICK scheme, on the contrary, predicts fairly well the magnitude of ejections~~ On the contrary, the FV solver with QUICK scheme underpredicts all the profiles except for the ejections, which are captured fairly well instead (see Fig. 7,a) ~~whereas profiles from the other quadrants are consistently underpredicted when compared to corresponding PSFD ones. It is well known.~~ Note that ejections are violent events, concentrated over a very thin region in the cross-stream direction of the ABL (Fang and Porté-Agel, 2015).
 300

To gain insight on the vertical structure of momentum transfer mechanisms, the exuberance ratio and the ratio of sweeps to ejections are analyzed in the following. Fig. 8(a) shows the exuberance ratio, defined as the ratio of negative to positive contributions to the momentum flux, $(S_{1,0} + S_{3,0}) / (S_{2,0} + S_{4,0})$ (Shaw et al., 1983). ~~Across the whole surface layer except~~
 305 ~~for the first node at the wall, the~~ The exuberance ratios from the PSFD-based solver are larger (in absolute value) than the

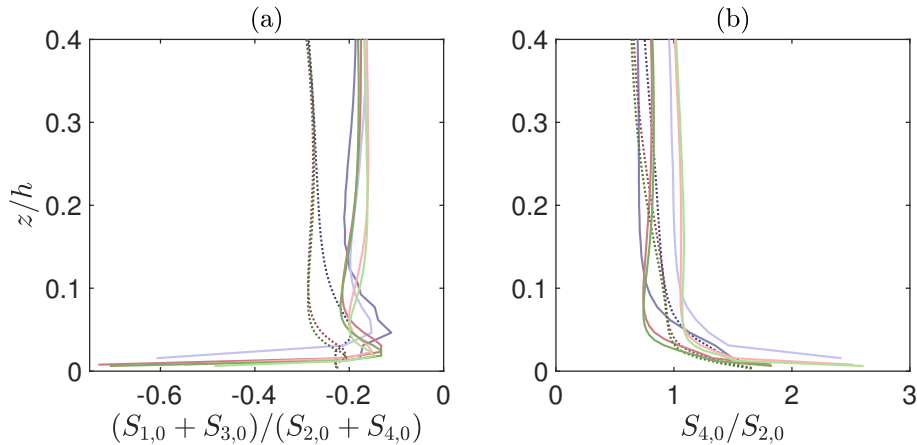


Figure 8. Stress fractions at $z/h \approx 0.1$. The profiles are normalized so that the sum of vertical structure of event ratios: (a) ratio of negative to positive contributions to the stress fractions for $H=0$ is unity across the cases; (b) ratio of sweeps to ejections. Lines are defined as in Fig. 1.

correspondent ones from the FV-based solver. The exuberance ratio profiles support findings from Fig. 7, which highlighted across the whole surface layer, except very close to the surface. Profiles highlight that outward and inward interactions have a significant impact on the resolved Reynolds stress in the PSFD-based solver, whereas the flow simulated with the FV-based solver is characterized predominantly by by a predominance of sweeps and ejections. This behavior is consistent throughout the ABL. Fig. 8(b) shows the ratio of sweeps to ejections in the lower at the lowest portion of the ABL ($z/h \leq 0.4$). Profiles obtained with the QUICK scheme are in line with predictions from the PSFD-based solver and with findings from measurements of surface-layer flow over rough surfaces, where ejections are identified as the dominant momentum transport mechanism in the ABL (Raupach et al., 1991). On the contrary, the FV-based solver with linear scheme tends to favor sweeps over ejections as the mechanisms for momentum transfer in the surface layer, throughout the considered grid resolutions.

To conclude the analysis on the mechanisms responsible for momentum transfer, the velocity statistics from a conditionally-averaged flow field is are discussed next. The approach of Fang and Porté-Agel (2015) is adopted to compute the conditionally-averaged flow field, where the conditional event is a positive streamwise velocity fluctuation u''/u_* at $\Delta x/h = 0$, $\Delta y/h = 0$, $z/h = 0.5$. Figure 9 features a pseudocolor pseudo-color and vector plot of the conditionally-averaged velocity field in a cross-stream-vertical plane for selected cases, whereas Fig. 10 displays a three-dimensional iso-surface thereof.

The flow structure in the equilibrium surface layer is known to be characterized by counter-rotating rolls and low- and high-momentum streamwise-elongated streaks flanking each others in the cross-stream direction. Rolls and streaks are indeed the dominant flow mechanism responsible for tangential Reynolds stress (Ganapathisubramani et al., 2003; Lozano-Durán et al., 2012).

As apparent from Fig. 9, the PSFD conditionally-averaged velocity field exhibits counter-rotating patterns associated with positive and negative streamwise velocity fluctuations (corresponding to the aforementioned streaks). The Throughout the

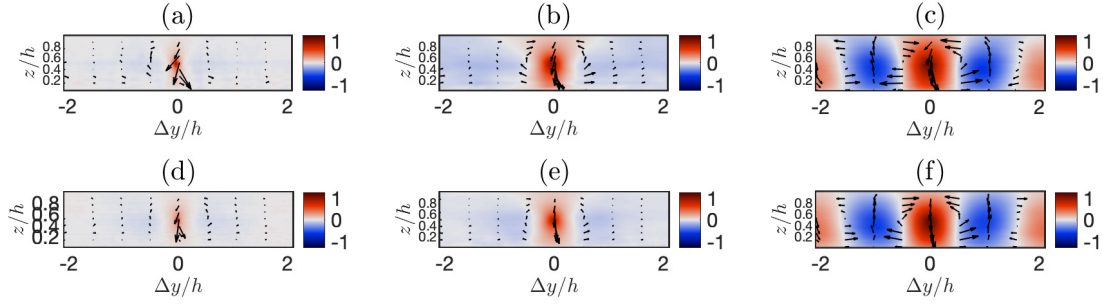


Figure 9. Vertical structure visualization of event ratios: the conditionally-averaged velocity field in the cross-stream-vertical plane at $\Delta x/h = 0$ from simulations FV64 (a) ratio of negative to positive contributions to the momentum flux, FV64* (b) ratio of sweeps to ejections, PSFD64 (c), FV160 (d), FV160* (e) and PSFD160 (f). Lines as in Fig. 9. The conditional event is a positive streamwise velocity fluctuation at $\Delta x/h = 0$, $\Delta y/h = 0$, and $z/h = 0.5$. Colors are used to represent the magnitude of the streamwise component, vectors denote the cross-stream and vertical components.

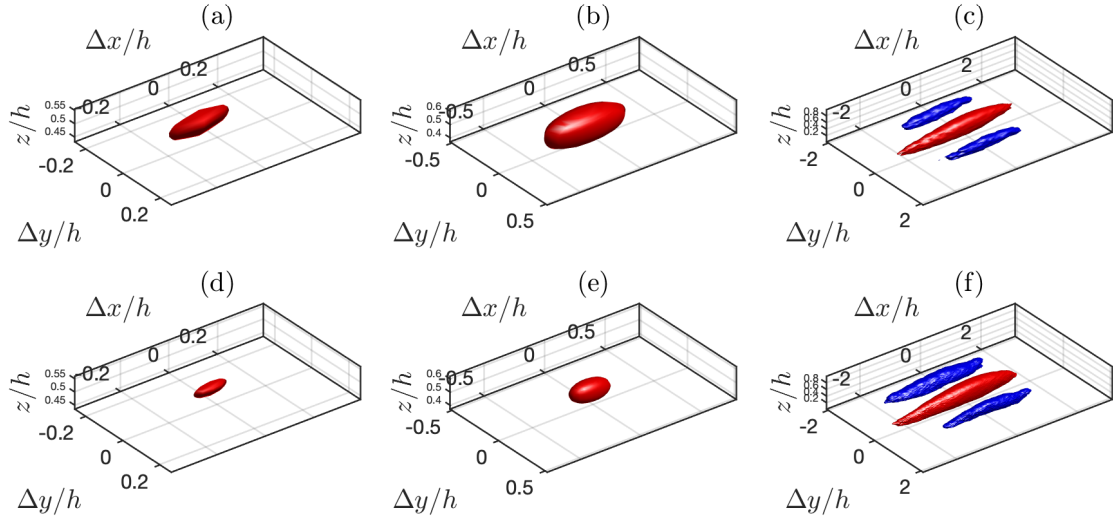


Figure 10. Conditionally-averaged flow field from simulations FV64 (a), FV64* (b), PSFD64 (c), FV160 (d), FV160* (e) and PSFD160 (f). The conditional average is computed as in Fig. 9. Red iso-surfaces show positive fluctuations (> 0.7 , top, and > 0.65 , bottom); blue iso-surfaces show negative fluctuations (< -0.55 , top, and < -0.5 , bottom).

ABL, the roll modes feature a diameters ($d \approx h$) throughout the ABL, diameter which is consistent with findings from the literature, and positive and negative velocity fluctuations are approximately of the same magnitude ($\approx u_\tau$) ($\approx u_*$). From Fig. 10, it is also apparent how the considered isosurfaces extend in apparent that the considered iso-surfaces extend about $4h$ along the streamwise direction for about $4h$. Quite surprisingly, the FV-based solver is not able to

330 predict the roll modes, irrespective of the interpolation scheme ~~or~~ and grid resolution, and severely underpredicts the magnitude of the low-momentum streaks ~~is also severely underpredicted across the considered cases~~. Further, Figs. 9 and 10 both depict a FV conditionally-averaged flow field that is poorly correlated in along the cross-stream and streamwise directions, resulting in significantly smaller momentum-carrying structures. This fact supports previous findings from the ~~two-point correlation maps~~ two-dimensional spatial autocorrelation (Fig. 4).

335 The lack of roll modes implies that ~~these solvers~~ the FV-based solvers here used are not able to capture the fundamental mechanism supporting momentum transfer in the ABL, at least at the considered grid resolutions. This limitation ~~can also be identified as is likely to be~~ the root cause of several of the observed problematics ~~with the FV solutions~~ associated with the FV-solver solution, including the relatively high (low) streamwise-velocity skewness when using using-linear (QUICK) schemes (see Fig. 2,a) and the ~~imbalance~~ observed imbalance between sweeps and ejections (Fig. 1 and Fig. 8).

340 ~~Visualization of the conditionally-averaged velocity field in the cross-stream-vertical plane at $\Delta x/h = 0$ from simulations FV64 (a), FV64* (b), PSFD64 (c), FV160 (d), FV160* (e) and PSFD160 (f). Colors are used to represent the magnitude of the streamwise component, vectors denote the cross-stream and vertical components. The conditional average is computed as in Fig. 10.~~

345 ~~Conditionally-averaged flow field from simulations FV64 (a), FV64* (b), PSFD64 (c), FV160 (d), FV160* (e) and PSFD160 (f). The conditional event is a positive streamwise velocity fluctuation $u''/u_* > 0.7$ at $\Delta x/h = 0$, $\Delta y/h = 0$, and $z/h = 0.5$. Red iso-surfaces show $u''/u_* > 0.7$ (top) and $u''/u_* > 0.65$ (bottom); blue iso-surfaces show $u''/u_* < -0.55$ (top) and $u''/u_* < -0.5$ (bottom).~~

4 Conclusions

350 ~~This~~ The present work provides insight on ~~the~~ quality and reliability of an important class of general-purpose, second-order accurate FV-based solvers in for the wall-modeled LES of neutrally-stratified ABL flow. The ~~FV~~ considered FV-based solvers are part of the OpenFOAM[®] framework, make use of the divergence form ~~of~~ for the nonlinear term, and are based on a collocated ~~arrangement for the evaluation of the unknowns~~ grid arrangement.

355 A suite of simulations ~~has been was~~ carried out in an open-channel flow setup, varying ~~grid resolution (with grid refinement the grid resolution~~ up to 160^3 control volumes), interpolation schemes for the discretization of the nonlinear term (~~linear and QUICK~~), the value of the Smagorinsky coefficient (~~$C_S = 0.1$ to $C_S = 1.678$~~), the pressure-velocity coupling method, and the time-advancement scheme (~~PISO with a second order Adams-Moulton and a projection method with a fourth order Runge-Kutta time stepping scheme~~). Several flow statistics ~~have been were~~ contrasted against profiles from a well established PSFD-based solver and against experimental measurements (~~when the latter were available~~) when these were available. Considered flow statistics include ~~the~~ mean velocity, turbulence intensities, velocity skewness and kurtosis, velocity spectra and spatial autocorrelations. An analysis of mechanisms supporting momentum transfer in the flow field ~~has also been proposed. Main was also proposed. The main~~ findings are summarized below.

With the exception of the FV solver ~~based on the projection and Runge-Kutta time advancement~~ with the projection method and the Runge-Kutta time advancement scheme, mean velocity profiles from the PSFD and FV solvers all feature a positive LLM ~~and existing~~. Existing techniques to alleviate this limitation ~~lead led~~ to no apparent improvements. This observation suggests that, for this class of solvers, alternative approaches should be devised to overcome this limitation in ABL flow simulations improvement, thus calling for alternative approaches.

Near-surface streamwise velocity fluctuations are consistently overpredicted by both the PSFD and FV solvers, irrespective of the grid resolution. The overshoot is particularly pronounced for the cases based on the QUICK interpolation scheme. This behavior can be related to a deficit of pressure redistribution in the budget equations for the velocity variances, which results in a pile-up pile-up of shear-generated streamwise velocity fluctuations and deficit in the vertical and cross-stream velocity fluctuation components.

The interpolation scheme used for the discretization of the nonlinear term plays a role in determining the remaining flow statistics. Specifically, FV solvers using with a linear interpolation scheme lead to ~~i)~~ -5pt a positive streamwise velocity skewness throughout the surface layer, which is at odds with experimental findings; ~~ii)~~ a severe overprediction of the streamwise velocity kurtosis; ~~iii)~~ a poorly correlated streamwise velocity field in the horizontal directions, especially at high grid resolutions; ~~iv)~~ a severe underprediction of outward and inward interactions and ejection events; ~~and v)~~ a lack of organized high- and low-momentum streaks ~~paired with and associated~~ roll modes in the conditionally-averaged flow field.

Grid resolution is either not affecting the above quantities or leading to larger departures from the ~~reference profiles~~. ~~When the QUICK scheme is used, the i) streamwise velocity skewness is better predicted when compared to the linear scheme and profiles show a convergence towards reference experimental measurements; ii) the kurtosis~~ expected behavior. The QUICK scheme, on the other hand, leads to -5pt an improved prediction of the streamwise velocity ~~is also better predicted~~ skewness and kurtosis, especially as the grid stencil is reduced; ~~iii) the a~~ streamwise velocity field ~~feature a higher degree of correlation when compared to those from the linear scheme in the that is more correlated along the~~ horizontal directions, but integral length scales ~~are still remain~~ only a fraction of those from the PSFD and reference DNS results; ~~iv) outward and inward interactions and sweep events are severely underpredicted; and v) there is again an underprediction of inward and outward interactions; a~~ lack of organized high- and low-momentum streaks and associated roll modes in the conditionally-averaged flow field.

To summarize, the considered class of FV-based solvers ~~overall~~ predicts a flow field that is less correlated ~~in space when compared that of the~~ than the one obtained with the PSFD solver and ~~is not able to do not~~ capture the salient ~~features mechanisms~~ responsible for momentum transfer in the ABL, at least at the considered grid resolutions. These limitations appear to be the root cause of many of the observed discrepancies between FV flow statistics and the ~~reference correspondent~~ PSFD or experimental ones, including the mispredicted streamwise-velocity skewness (Fig. 2,a), the ~~inbalance imbalance~~ between sweeps and ejections (Fig. 1 and Fig. 8), and the overall sensitivity of flow statistics to variations in the grid resolution.

395 Findings from this study indicate that higher grid resolutions—or a different arrangement of the computational grid—might be required to correctly capture the aforementioned quantities and achieve resolution-independent results in wall-modeled LES of neutrally-stratified ABLs. Given Higher grid resolutions might help alleviate some of these shortcomings, but given that grid resolutions used herein are state-of-the-art for general-purpose general-purpose FV-based solvers and that computing power increases relatively slowly with time (Moore, 1965), the aforementioned limitations are likely to persist for years to come and introduce, thus introducing a degree of uncertainty in model results that needs to be addressed. This calls for further These limitations call for research aimed at reducing the impact of discretization and modeling errors, such as introducing discretizations that rely errors in this class of solvers, or for alternative approaches such as leveraging discretizations based on staggered grid arrangements or using and higher-order spatial-discretization spatial discretization schemes.

– Code availability. OpenFOAM® is an open-source computational fluid dynamics toolbox. The present study made use of OpenFOAM®
405 version 6.0, available for download at <https://openfoam.org/version/6/>. Data and scripts to generate figures in this manuscript can be downloaded from <https://gitlab.com/turbulence-columbia/miscellaneous/fv-solvers-abl-flow>.

Appendix A

A1 Smagorinsky constant

A sensitivity analysis on We here test the sensitivity of selected flow statistics to variations in the Smagorinsky constant C_S
410 is here performed. In addition to $C_S = 0.1$, the values $C_S = 0.12$, $C_S = 0.14$, $C_S = 0.16$, and $C_S = 0.1678$ C_S . The values $C_S = 0.1$, $C_S = 0.12$, $C_S = 0.14$, $C_S = 0.16$, and $C_S = 0.1678$ (the default value in OpenFOAM®) are considered. All the tests are run on, and all tests are carried out at 64^3 control volumes.

As shown in Fig. A1(a), the Smagorinsky constant has an a relatively important and non-monotonic impact on the mean
streamwise-velocity profile. The case at $C_S = 0.1$ $C_S = 0.1$ results in the largest positive LLM, in agreement with the pre-
415 dictions from the PSFD-based solver, whereas the cases at larger C_S C_S exhibit a smaller, albeit still positive, LLM. The Smagorinsky coefficient has also a discernible impact on the velocity RMSs. Specifically, as C_S is increased, the magnitude of the near-surface maximum for both u'_{RMS} (Fig. A1,b) and w'_{RMS} (Fig. A1,c) is reduced, and the location of the maximum is shifted away from the surface—possibly the result of a higher near-surface energy dissipation as C_S is increased. In addition, larger values of C_S C_S yield a more apparent departure from the corresponding profiles obtained with the PSFD-based solver.

420 The one-dimensional spectra (Fig. A2,a) show that larger values of the Smagorinsky coefficient result in a more rapid decay of energy density and in a shift of profiles toward the inertial sub-range. Interestingly, all the profiles exhibit the same slope, hence the same power-law exponent. No value of the Smagorinsky coefficient seems suitable for capturing the $k^{-5/3}$ k^{-1} power law in the inertial sub-range. Increasing C_S production range of turbulence. Increasing C_S leads to a modest improvement in the profiles of the spatial autocorrelation two-point autocorrelation profiles (Fig. A2,b and Fig. A2,c).

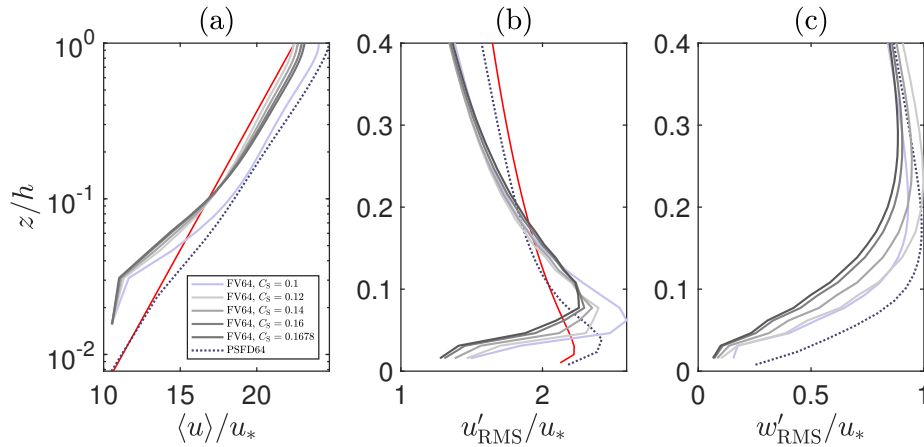


Figure A1. Vertical structure of streamwise velocity (a), streamwise velocity RMS (b), vertical velocity RMS (c). Red lines denote the phenomenological logarithmic-layer profile (a) and the analytical expressions from similarity theory (Stull, 1988) (b, c).

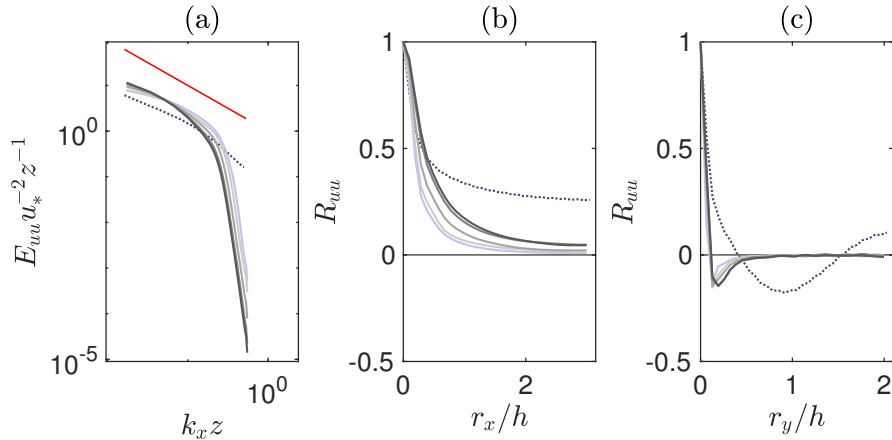


Figure A2. Normalized one-dimensional spectrum spectra of streamwise velocity at height $z/h \approx 0.1$ (a); one-dimensional spatial autocorrelation of streamwise velocity at height $z/h \approx 0.1$ along the streamwise direction (b) and along the cross-stream direction (c). Lines as in Fig. A1. Red line, $(k_x z)^{-1}$.

425 A2 Solvers

In this Sub-Appendix, The performance of an alternative solver within the OpenFOAM[®] framework is here considered, and the results are contrasted against those previously shown (obtained with the PISO algorithm in combination with an Adams–Moulton time-advancement scheme). The solver is based on a projection method coupled with the Runge–Kutta 4 time-advancement scheme (Ferziger and Peric, 2002). Details on the implementation can be found in Vuorinen et al. (2015) (note that, in their reported code, a term in the form of a time step Δt is missing, leading to a dimensional mismatch and raising a

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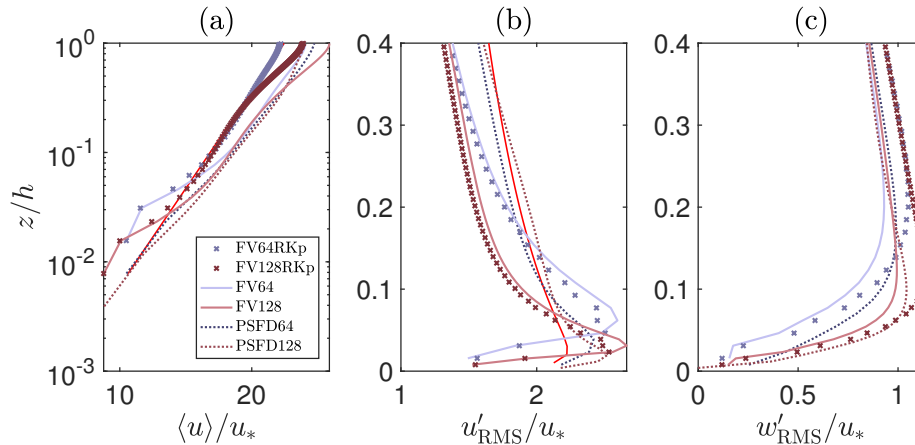


Figure A3. Vertical structure of streamwise velocity (a), streamwise velocity RMS (b), vertical velocity RMS (c). Red lines denote the phenomenological logarithmic-layer profile (a) and the analytical expressions from similarity theory (Stull, 1988) (b, c).

~~compile-time error~~). The performances of the two solvers ~~have already been are~~ compared at moderate Reynolds number in Vuorinen et al. (2014), where it is pointed out that the projection method coupled with the Runge–Kutta 4 time advancement scheme provides similar results at lower computational cost. In the following, the performances of the solver are tested ~~at high Reynolds number ($Re_\tau = 10^7$)~~. ~~Two simulations, over~~ for the considered ABL flow. Two grid resolution are considered, based
 435 on 64^3 eubes (case FV64RKp) and over 128^3 eubes (case FV128RKp) , are considered control volumes.

~~In Fig. A3(a) the The~~ vertical profile of the mean streamwise velocity is shown ~~in Fig. A3(a)~~. The use of the projection Runge–Kutta 4 solver leads to an underprediction of the velocity at the wall as for the simulations FV64 and FV128. ~~Interestingly, the profiles feature no, but no apparent~~ LLM in the surface layer. ~~The streamwise and vertical velocity RMSs, shown in u'_{RMS} exhibits the previously observed near-surface peaks (Fig. A3(b) and b) whereas w'_{RMS} is overpredicted above~~
 440 $z/h = 0.15$ (Fig. A3(c), exhibit the behavior already analyzed in §3.1, with an underprediction in the near-wall region, an overprediction of the u'^{\pm} peak and an underprediction of the w'^{\pm} peak, c).

Author contributions. BG and MG designed the study, BG conducted the analysis under the supervision of MG, BG and MG wrote the manuscript.

Competing interests. The authors declare that they have no conflict of interest.

445 *Acknowledgements.* The work was supported via start-up funds provided by the Department of Civil Engineering and Engineering Mechanics
at Columbia University. The authors acknowledge computing resources from Columbia University's Shared Research Computing Facility
project, which is supported by NIH Research Facility Improvement Grant 1G20RR030893-01, and associated funds from the New York State
Empire State Development, Division of Science Technology and Innovation (NYSTAR) Contract C090171, both awarded April 15, 2010.
The authors are grateful to Weiyi Li for generating the PSFD data, and to Drs. Ville Vuorinen and George I. Park for useful discussions on
450 the performance of finite-volume based solvers for the simulation of turbulent flows.

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